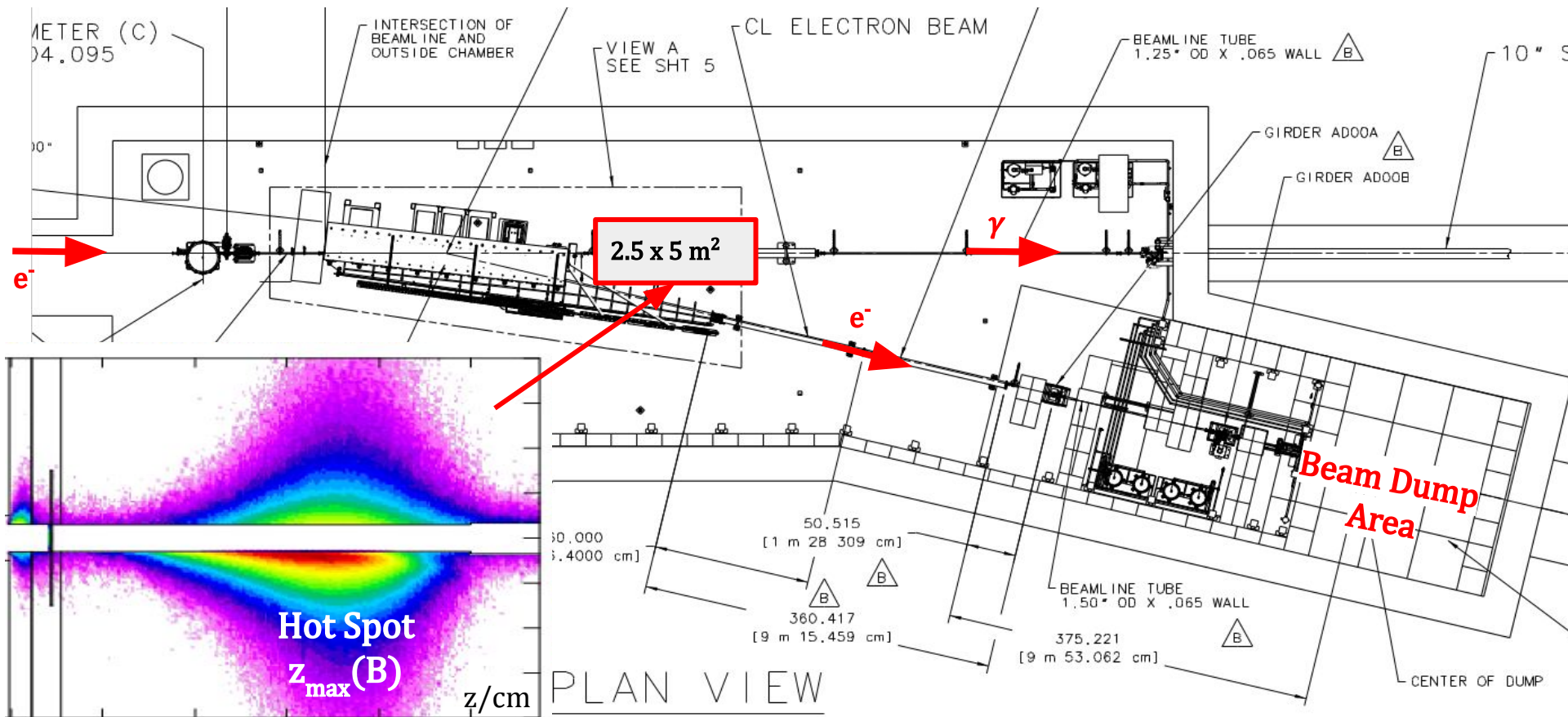


# CPS with 3 m long Dipole in the Tagger Hall D.

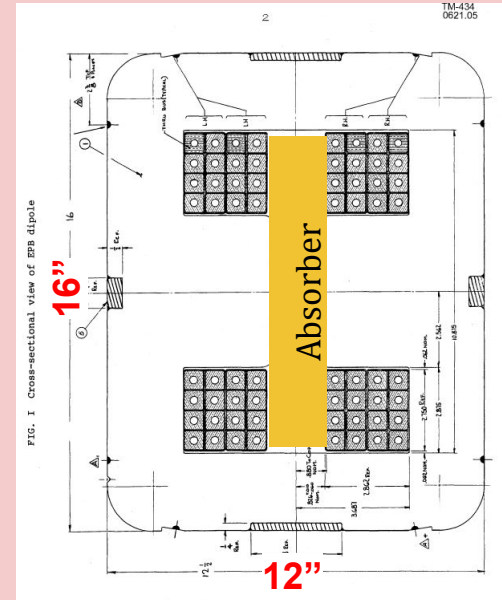
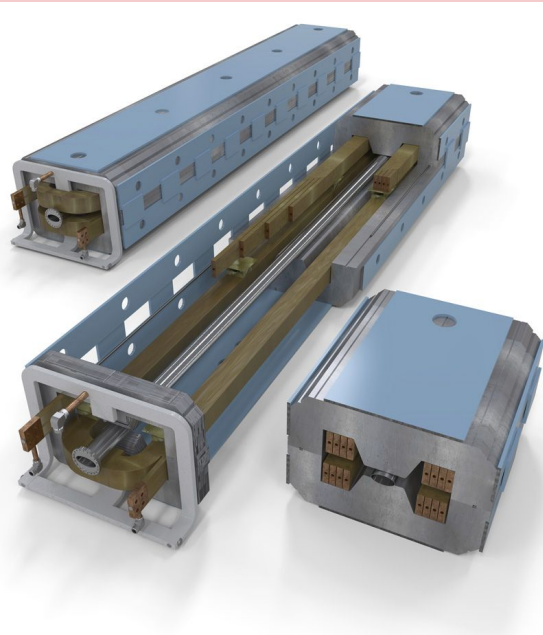




# CPS 3 m Long Dipole.

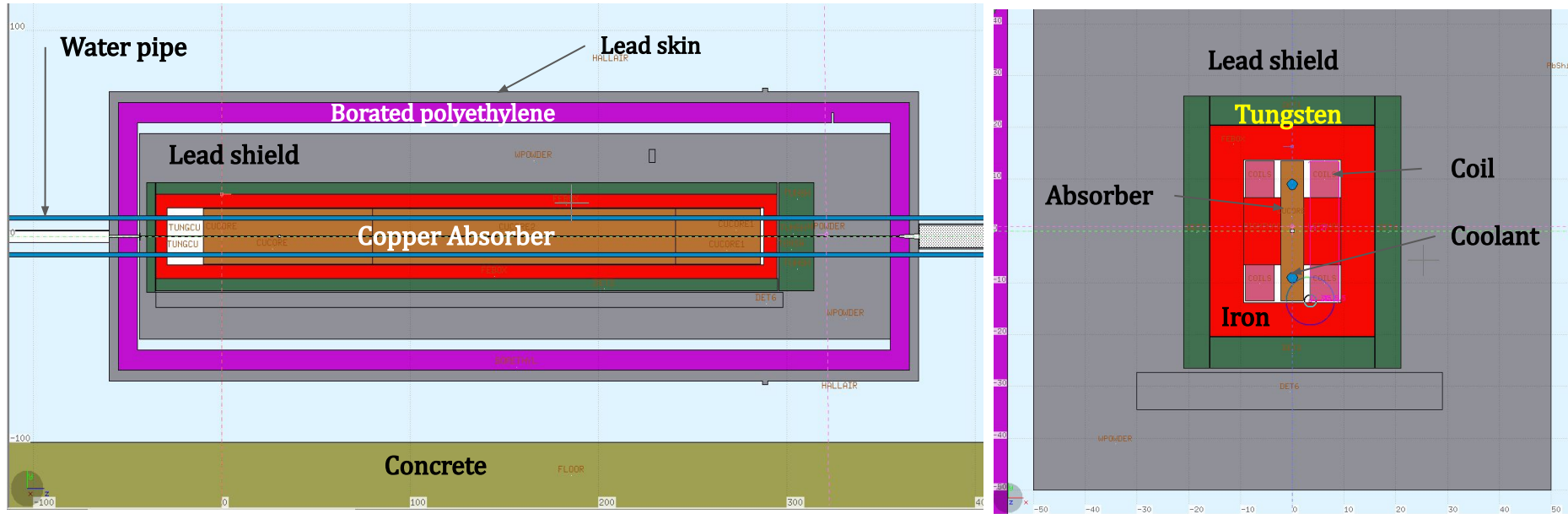
FNAL prototype 1.5 T  $\times$  3 m, pole gap 4.2 cm.

1. FLUKA model.
2. Hot Spot Size and Magnetic field.
3. Coil Insulation Lifetime.
4. Photon Beam Quality and Coil.
5. Radiological safety.
6. Absorber  $T^{\circ}$  and cooling.



# FLUKA model with FNAL prototype operating at $B \sim 0.1$ T.

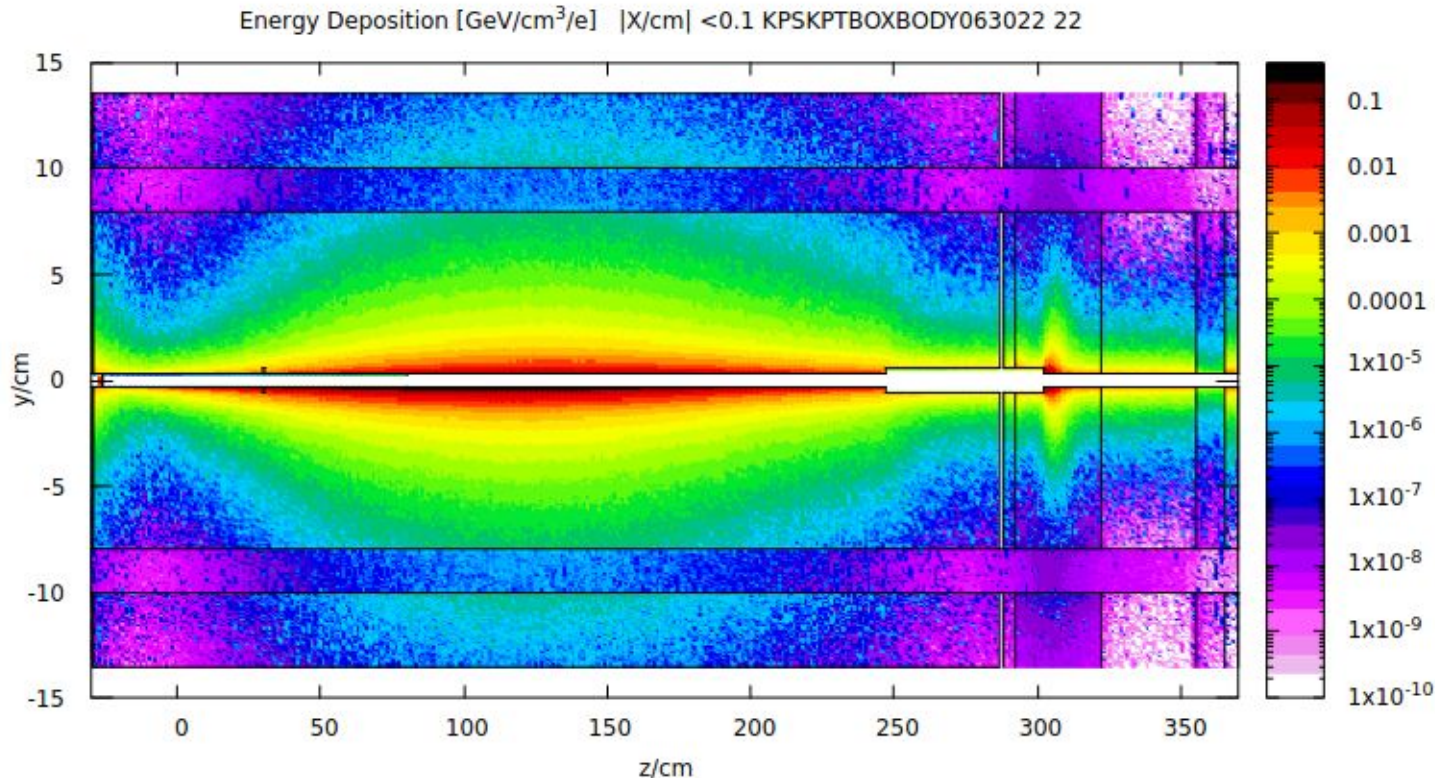
e-Beam  $5 \mu\text{A}$ , FWHM = 0.25 cm, e-Beam Hole =  $0.6 \times 0.6 \text{ cm}^2$ , Cu radiator 0.134 cm.



- **Absorber design** (max.  $T^0$ ) with autonomous cooling **determines Dipole dimensions.**
- **Low field** allows wider gap between poles  $\Rightarrow$  **More room** for cooling pipes.
- Magnet coil may be **0.4 m long**  $\Rightarrow$  more room for Absorber in Hot Spot.

# Cooling pipe location and Energy Deposition in Absorber. Example at B=0.22 T.

e-Beam 5  $\mu\text{A}$ , FWHM = 0.25 cm, e-Beam Hole =  $0.6 \times 0.6 \text{ cm}^2$ , Cu radiator 0.134 cm.

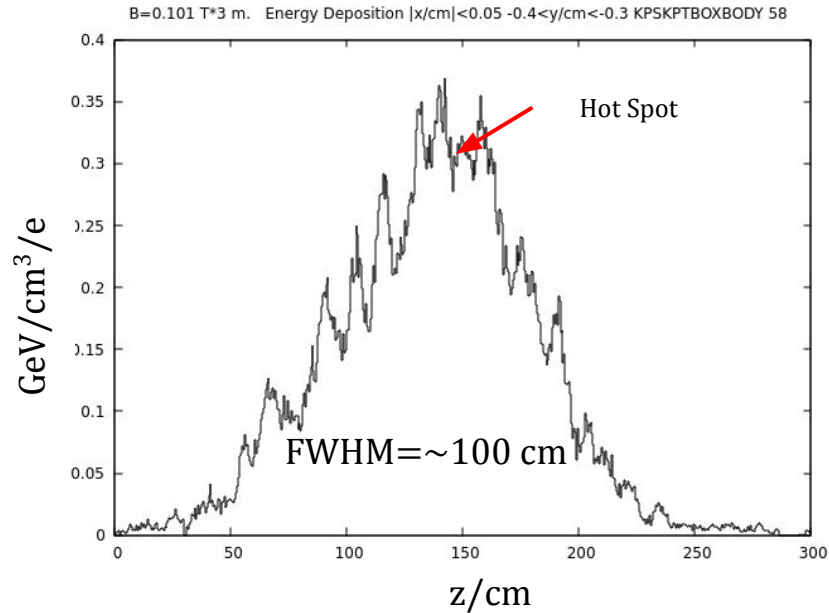


- Inside water pipes  $P_{\text{max}} = 2.E-6 [\text{GeV}/\text{cm}^3/\text{e}] \times 3.E+13 [\text{e}/\text{s}] \times 1.6 E-10 [\text{J}/\text{Gev}] = 0.01 [\text{W}/\text{cm}^3]$ .
- Total  $\sim 12 \text{ W}/\text{pipe}$ . Looks like there is **no** problem with **water overheating**.

Power Deposition in Hot Spot vs Magnetic Field.

Coil 3 m long vs Coil 0.4 m long.

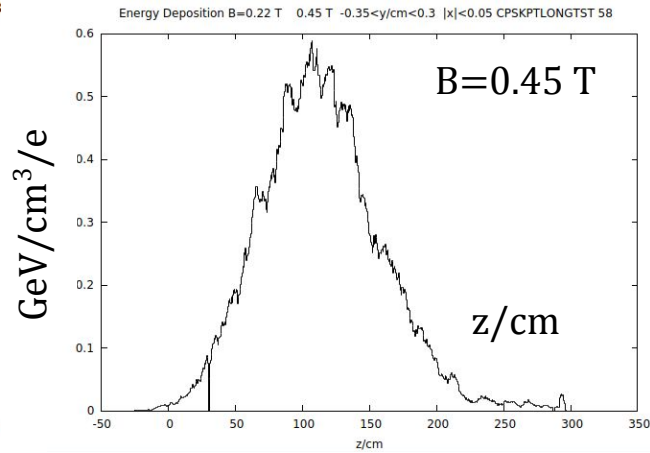
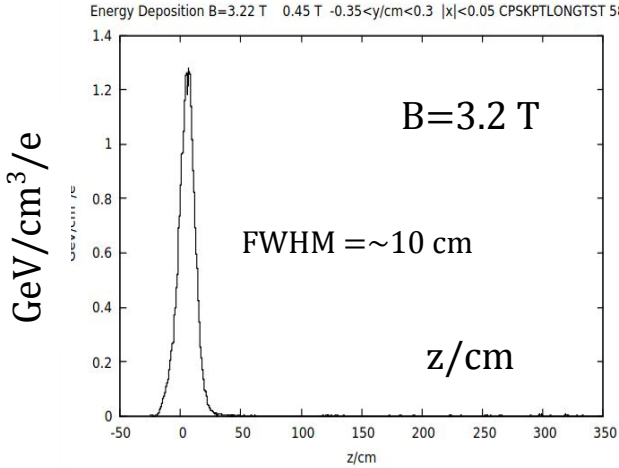
# Power Deposition in Hot Spot . Coil 3 m long B=0.1 T.



- 45 kW are deposited in the Absorber.
- Maximum power deposition in the Hot Spot:  $dE/dzdt = 45 \text{ kW}/FWHM = 0.45 \text{ kW/cm}$ .
- This value determines the **Hot Spot Temperature**.

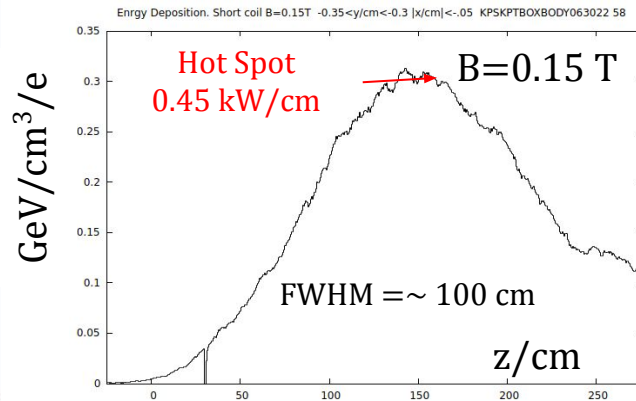
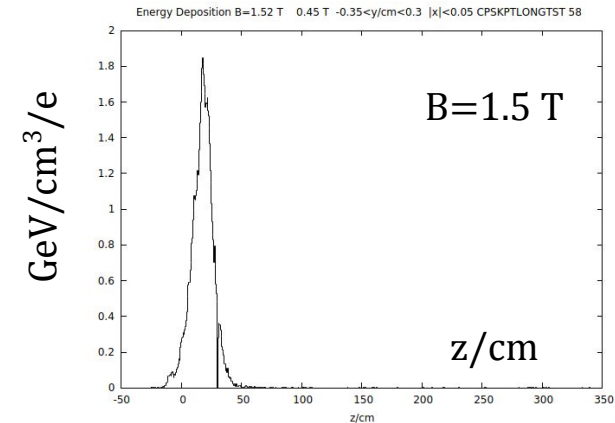


# CPS with 0.4 m coils. Energy deposition profile vs Magnetic Field.



- If 45 kW are deposited in the Absorber, the maximum power deposition

$$dE/dzdt = 45 \text{ kW}/\text{FWHM} = \mathbf{0.45 \text{ kW/cm.}}$$



- Hot Spot  $T^0 \propto \max dE/dzdt.$

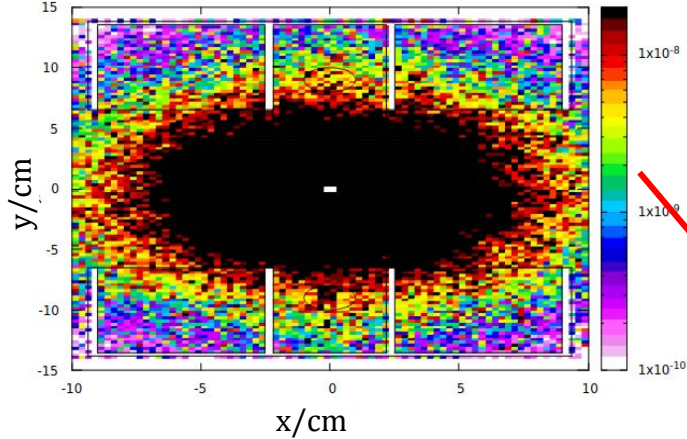
Lifetime of Coil Insulation  
VS  
Coil Design.



# Short coil vs Long coil. Prompt Dose and Insulation Lifetime.

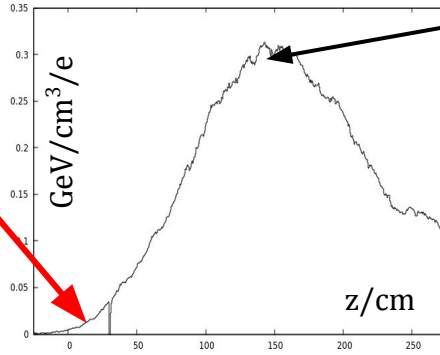
Dose in 40 cm coil at  $10 < z/\text{cm} < 15$

Dose upst. coil (-25 < z/cm < 15) 10 < z/cm < 15 black=2.e-8 [GeV/g/e] KPSKPTBOXBODY063022 28



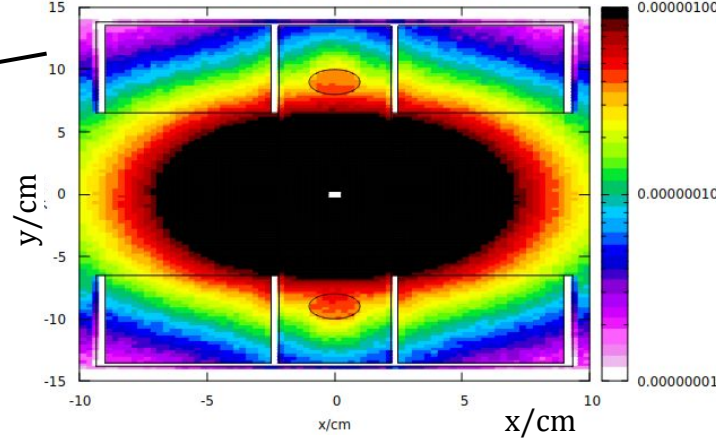
Energy Dep. z-profile

Energy Deposition. Short coil B=0.15T -0.35 < y/cm < 0.3 |x/cm| < .05 KPSKPTBOXBODY063022 58



Dose in 3m coil at  $145 < z/\text{cm} < 155$

Dose middle. coil 145 < z/cm < 155 black=1.e-6 [GeV/g/e] KPSKPTBOXBODY063022 29

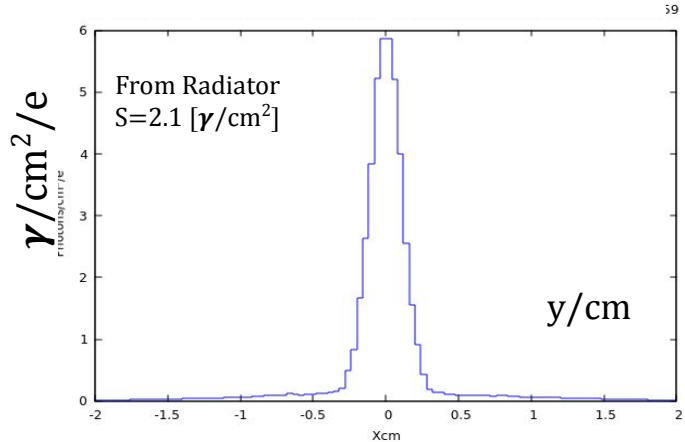


- **Short 40 cm coil** is wind within  $-25 < z/\text{cm} < 15$  ; Long **3m coil** – within  $-28.5 < z/\text{cm} < 287$ .
- The maximum dose in the **40 cm coil**  $\sim 2.E-8$  [GeV/g/e] or  $3.2E-15$  [Gy/e] ( $\times 1.6E-7$  Gy/(GeV/g)).
- At  $5 \mu\text{A}$  we have  $3.E+13$  [e/s]  $\Rightarrow$  in 40 cm coil dose rate =  $3.2E-15$  [Gy/e]  $\times 3.E+13$  [e/s]  $\cong 0.1$  [Gy/s].
- **Kapton withstands  $1.E+7$  [Gy]  $\Rightarrow$  Lifetime  $\sim 1.E+8$  [s] = **1160 days** of continuous operation.**
- In **3m coil** the maximum dose  $\sim 0.6E-6$  [GeV/g/e], and **Lifetime = 40 days** .
- Lifetime may be **5 times longer** using **fiberglass cloth**.

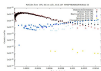
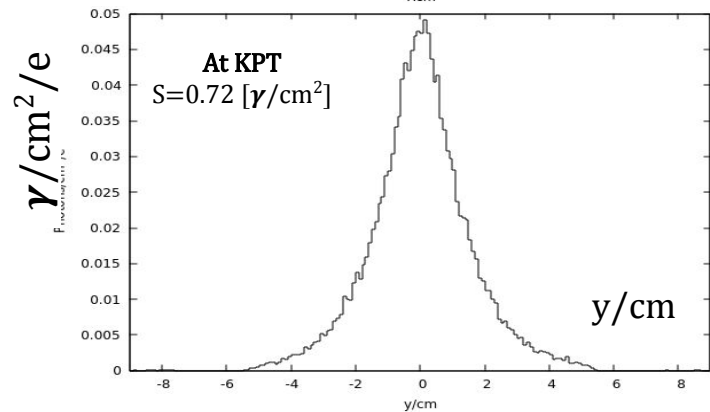
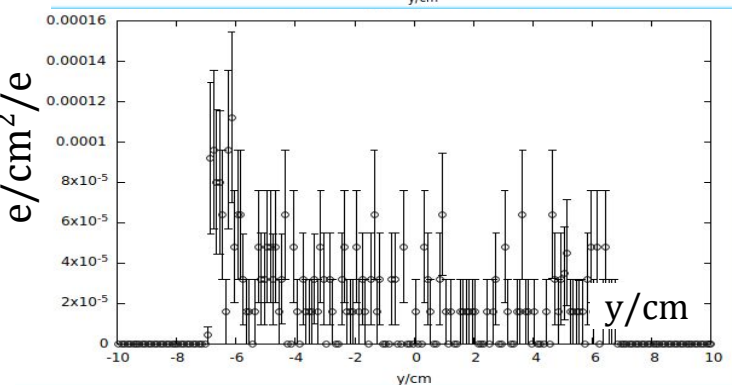
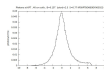
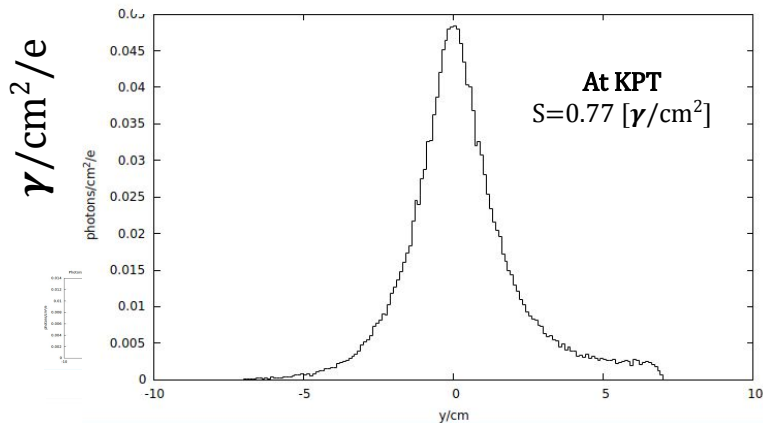
# Photon beam quality. Short Coil vs Long Coil.

# Photon Beam profile at KPT. Long vs Short coil.

## Coill 3 m B=0.1 T



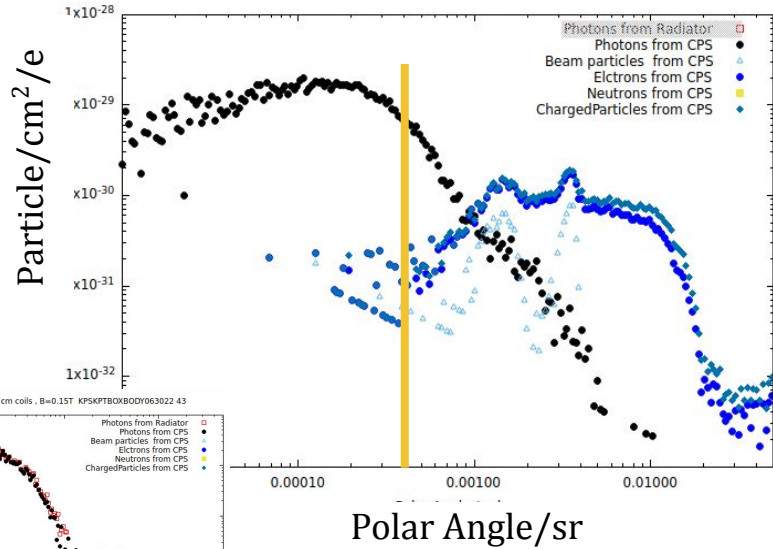
## Coill 0.4 m B=0.18 T



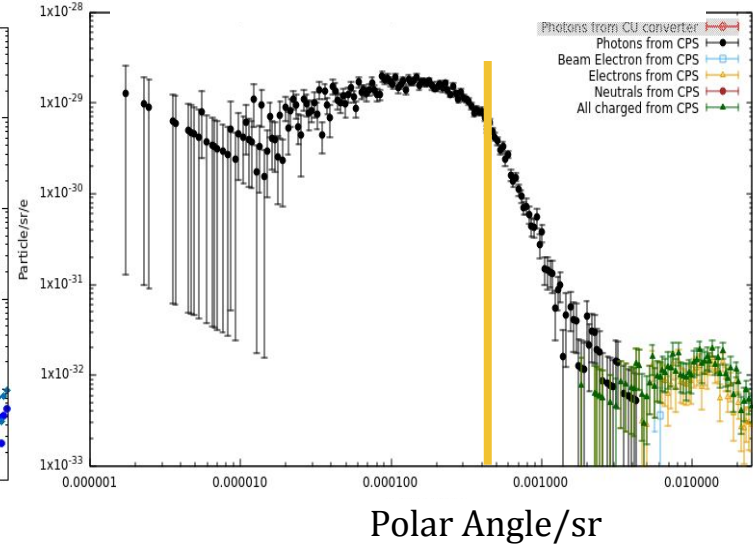
- Photon beam profile fits well the Be target of KPT, but there is  $\sim 1\%$  background of electrons.

# Photon Beam quality at CPS exit. Coil 0.4 m vs 3 m long.

## Coil 0.4 m B=0.18 T



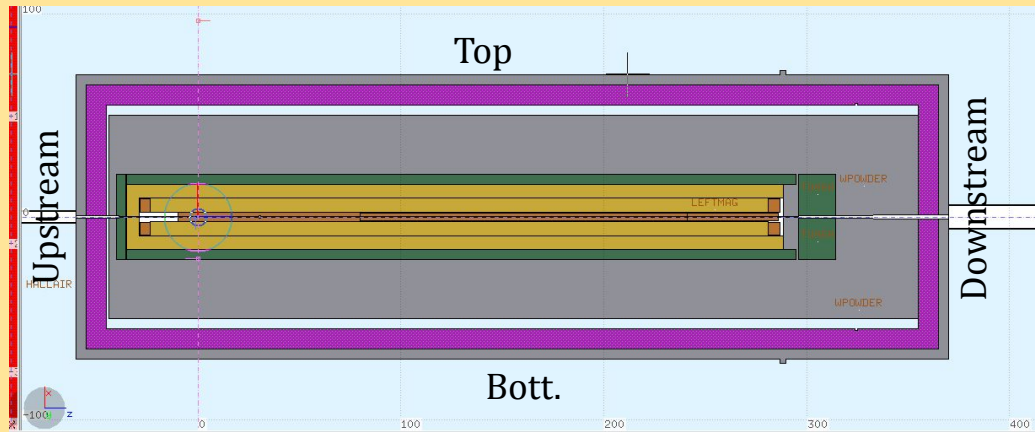
## Coil 3 m B=0.1 T



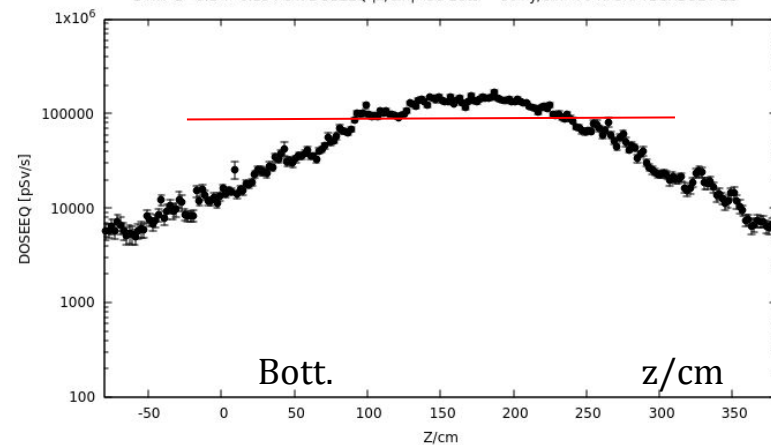
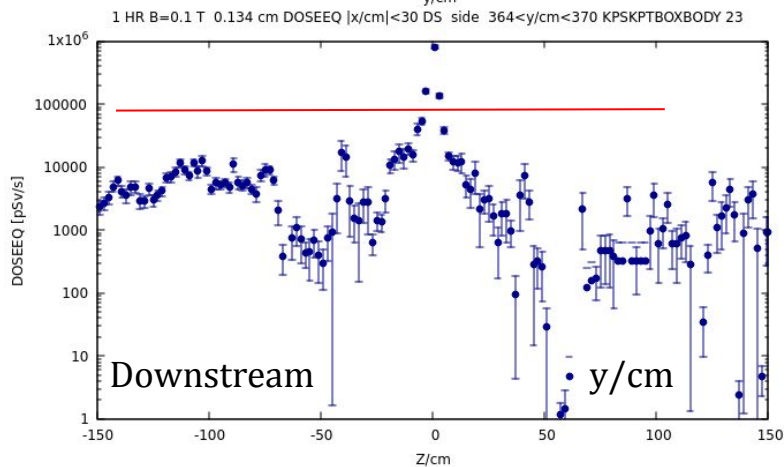
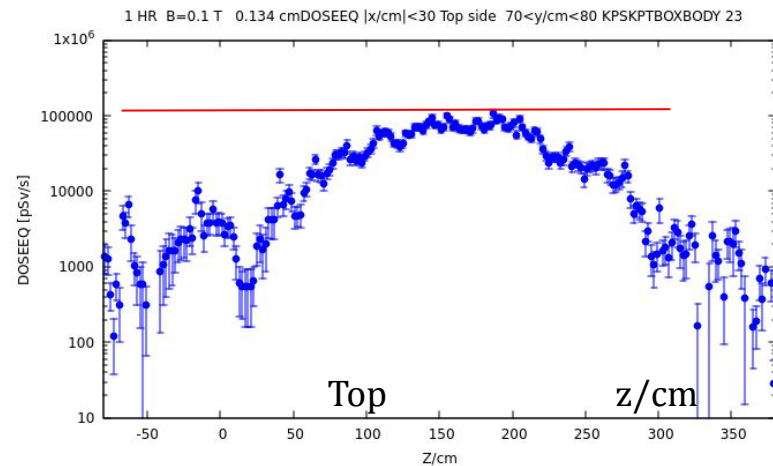
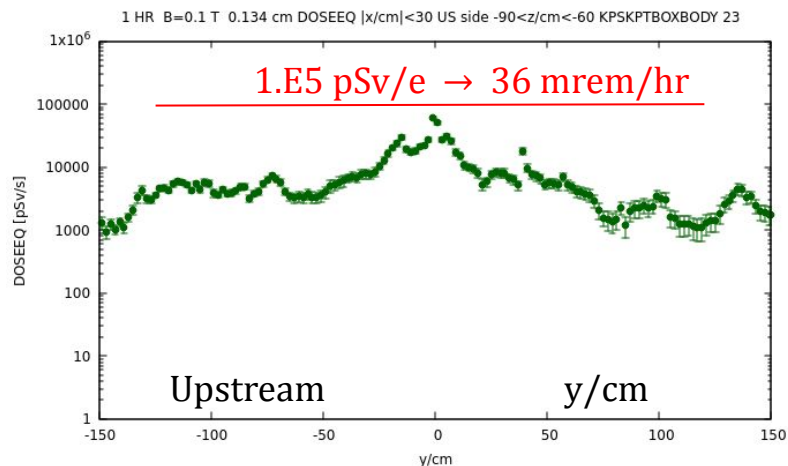
- Background is below 1% in the KPT acceptance of 0.4E-3 sr.
- May be cleaned up by the beam line **permanent magnet downstream** the CPS.

# Radiological safety.

Dose Equivalent rates after 1000 hrs of continuous operation and 1 hr break at 4 surfaces of CPS



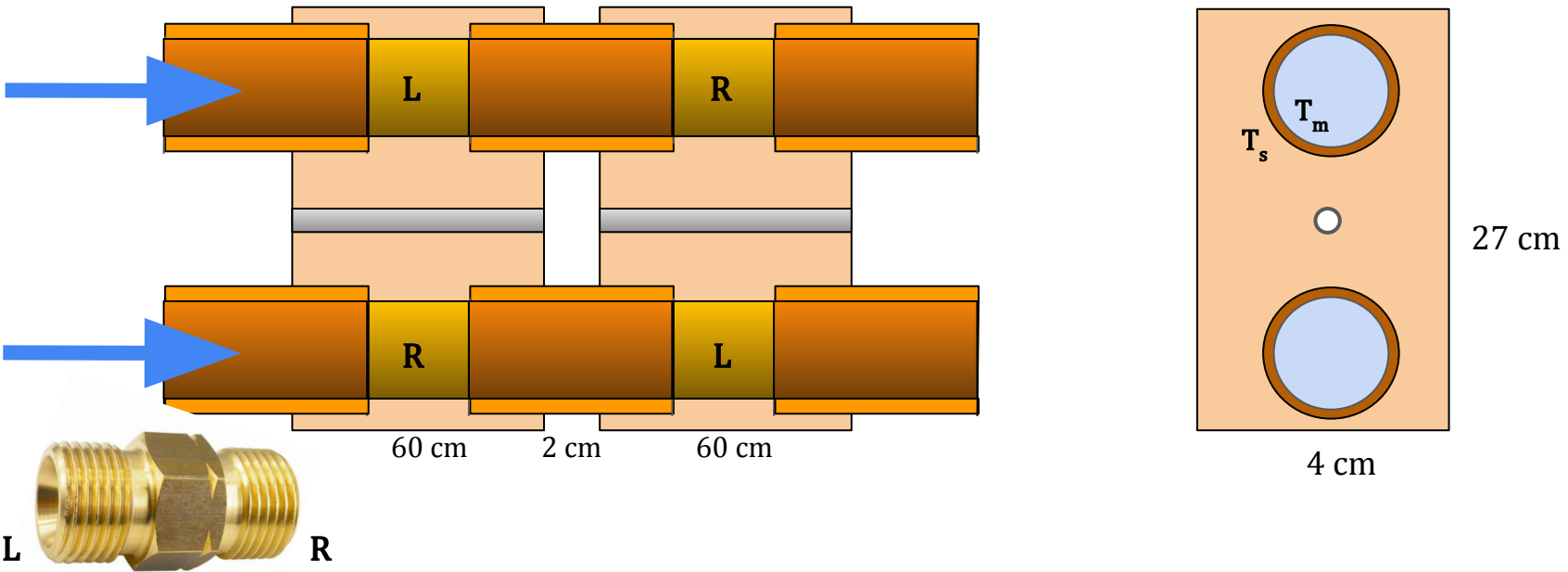
# After One Hr Dose Equivalent is below 100 mrem/hr . Dipole 3 m $\times$ 0.1 T.



# Absorber Design and Cooling

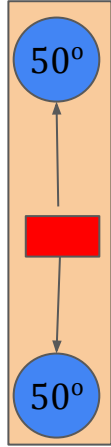


# Segmented Copper Absorber inside Dipole 0.1 T × 3 m.



- Absorber with **round beam hole** eliminates the problem of top-bottom thermal contact.
- **Direct contact to cooling liquid** in each segment is an advantage.

# Cooling and Temperature jump at the Absorber-Water boundary.



Copper Bar  
 $l=100$  cm  
 $w=4$  cm  
 $h=27$  cm.

- What should be the temperature difference between Absorber and Coolant ?
- For heat transfer rate from the surface at  $T_s$  to liquid at  $T_m$  via area  $A$  we write:

$$dQ/dt=k (T_s - T_m) A = 45 \text{ kW},$$

where area  $A= \pi w l =1250 \text{ cm}^2$  ,

$k=4.E-2 \text{ [W cm}^{-2} \text{ K}^{-1}]$  - “**heat transfer coefficient for laminar flow**”;

**tabulated empirical value** for “water-Cu-water” contact; to be doubled for “Cu-water” contact.

May be 2-3 times higher for **turbulent** flow. Is it **used in FEA calculations (?)**.

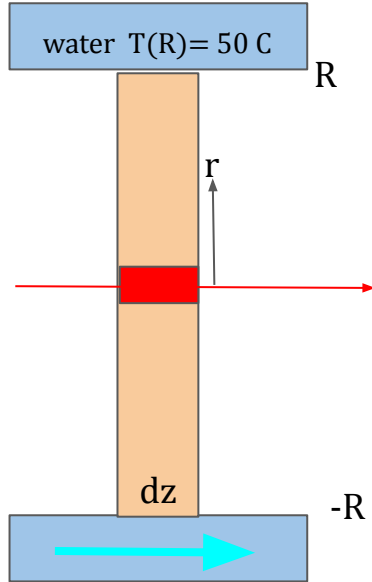
To sink the uniformly distributed 45 kW power from  $4 \times 27 \times 100 \text{ cm}^3$  bar, we find:

$$T_s - T_m = 45 \text{ kW} / (8. E-2 \text{ [Wcm}^{-2} \text{ K}^{-1}] 1.25 E+3 \text{ [cm}^2]) = 450 \text{ K}.$$

- **At  $T_m = 50 \text{ C}$  Absorber temperature at contact with water yields:  $T_s = 500 \text{ C}.$**

- To reduce  $T_s$  make Absorber width proportional to  $dE/dzdt$ .
- How **heat transfer coefficient** is addressed in **Fluence FEA software?**

# Absorber slice heating by 0.45 kW/cm in Hot Spot.



Copper Bar  
 $dz \times a \times 2R\text{ cm}^3$

- Energy deposition rate in the Absorber dz-slice along beam line :  

$$dE/dt = dz \times dE/dzdt = dz \times 0.45\text{ kW/cm};$$
- Energy balance in case of steady heat flow :  $dz (dE/dzdt) = dz (-2 a \kappa dT/dr)$ ,  

$$\Rightarrow dE/dzdt = -2a\kappa dT/dr,$$

$$dT = -(2a\kappa)^{-1} dE/dzdt dr, \quad \text{integrate } r < R,$$

$$T(r) - T(R) = (2a\kappa)^{-1} dE/dzdt (R - r),$$

where

$dE/dzdt = 0.45\text{ [kW/cm]}$  ; from previous slide.

$T(R) = 500\text{ C}$  ; estimate of Absorber temperature from the previous slide

Absorber height =  $2R$ ;  $R = 8\text{ cm}$ ; Absorber width  $a = 3.7\text{ cm}$

Heat conductivity for Copper  $\kappa = 3.98\text{ W/cm}\cdot\text{K}$  ; for Tungsten =  $1.73\text{ W/cm}\cdot\text{K}$ .

$T(0) - T(R) = (2 \cdot 3.7 \cdot 3.98)^{-1} [\text{K/W}] \cdot 0.45 \cdot 10^3 [\text{W/cm}] \cdot 8 [\text{cm}] = 120\text{ K}$ . ( $\sim 300\text{ K}$  for tungsten)

Maximal Hot Spot Temperature yields:  $T(0) = 500\text{ C} + 120\text{ K} = 620\text{ C}$ .

- Looks like we may avoid melting of copper.
- FEA calculation of the Absorber cooling are required ASAP using Fluence software.

# Is it possible to evacuate 60 kW from the Absorber?

- Water speed required to evacuate  $dE/dt=60$  kW through 3 cm-pipe  $L=3$  m,  $S=7$  cm<sup>2</sup>

$$dE/dt=C_v(T_{out}-T_{in}) v S, \quad (1)$$

where

$C_v = 4.2$  [JK<sup>-1</sup>cm<sup>-3</sup>] – specific heat capacity of water

$T_{out}-T_{in} = 70$  C – water flow temperature change.

With these numbers Eq. (1) yields:

$$6.E+4W = 4.2 [J K^{-1}cm^{-3}] 70 [K] 7 [cm^2] v [cm/s],$$

and we find  $v = 30$  [cm/s].

- Heating time is of 10 s that is enough to reach  $T_{out}=90$  C.
- Water flow speed looks consistent with the power deposition of 60 kW.
- Absorber design looks practical and may be optimized using FEA of heat flow.



## Conclusive remarks

- With **~0.4 m long coils and lower field ~0.1 T** we avoid the **risk of Absorber overheating**.
- With **~0.4 m coils and lower field** we avoid the **risk of coil short circuit**, for up to 5000 days.
- **After 1 hr Dose rates** at CPS surfaces are far **below 100 mrem/hr** (High Radiation Area).
- **Photon beam** may additionally **cleaned up** with the Hall D beam line magnet.
- Total **CPS weight** is of 60 metric tons.
- Long magnet yoke (3 m) is a stable and adjustable housing for Absorber.



## What we need to proceed with Dipole design.

- Thermal **FEA** model (Fluence software ) of Absorber and **cooling lines** with known *heat transfer coefficients*.
- **Mesh for thermal calculations** should scale in fraction of *mm* in the hot spot.
- Iterative **Temperature field** calculation using **Energy Deposition** Map from FLUKA to **optimize Dipole** dimensions.