## Sumpary of the Absorber Temperature Calculations

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## What Has Been Done?

- Several conditions have been simulated with FLUKA for each model.
- Quite a bit of effort from Vitaly and Pavel.
- The power deposition data have been analyzed with Mathematica
- I can use both rectangular mesh or cylindrical mesh.
- Cylindrical mesh is good when the axis is near the hot spot
- Relatively small mesh and faster analysis ( $\sim 2$-3hrs per condition).
- Problems with boundary conditions when they involve the temperature gradient.
- Workarounds and hacks are needed.
- Most likely has larger uncertainties than rectangular mesh with a fine element size for the solver.
- Rectangular mesh needs to have much finer element sizes near the hotspot.
- Larger mesh and longer run times (16-24hrs per condition)
- Simple to setup the model and the mesh.
- The sets were analyzed with cylindrical mesh in the solver.
- For Vitaly's data, I converted the cylindrical grid coordinates to Cartesian .
- Some of the outstanding settings were analyzed with rectangular mesh.
- The thermal analysis in complete
- Tim needs to check the static structural analysis in ANSYS to check for safety factors and margins.
"BC-65-m23" Test Summary (Vitaly)

| Test Configuration Name | $\mathrm{R}_{\max }(\mathrm{cm})$ | $\phi_{\max }(\mathrm{deg})$ | $\mathrm{Z}_{\text {max }}(\mathrm{cm})$ | $\mathrm{T}_{\max }\left({ }^{\circ} \mathrm{C}\right)$ | $\mathrm{T}_{\text {cold }}\left({ }^{\circ} \mathrm{C}\right)$ | Maximum power $\left(\mathrm{KW} / \mathrm{cm}^{3}\right)$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| All Nominal ( $\sigma^{(x, y)}{ }_{\text {beam }}=1 \mathrm{~mm}, 4$ holes) | 0.32 | -90 | 135 | 90 | 50 | 2.9 (Total 53 KW ) |
| $\sigma^{(x, y)}{ }_{\text {beam }}=0.33 \mathrm{~mm}$ | 0.32 | -90 | 140 | 135 | 55 | 8.0 (Total 53 KW ) |
| $\sigma^{(x, y)}{ }_{\text {beam }}=1.5 \mathrm{~mm}$ | 0.32 | -90 | 50 | 120 | 55 | 1.8 (Total 53 KW ) |
| 90\% B-field | 0.32 | -90 | 150 | 88 | 50 | 2.5 (Total 53 KW ) |
| 110\% B-field | 0.32 | -90 | 120 | 102 | 55 | 7.0 (Total 53 KW ) |
| -1mm shift in $Y$ | 0.33 | -90 | 50 | 145 | 55 | 4.2 (Total 54 KW ) |
| +1mm shift in $Y$ | 0.32 | -90 | 165 | 90 | 50 | 2.7 (Total 53 KW ) |
| -0.5mrad angle in $Y$ | 0.32 | -90 | 100 | 110 | 50 | 3.7 (Total 54 KW ) |
| +0.5mrad angle in $Y$ | 0.65 | -90 | 355 | 100* | 50 | 2.2 (Total 52 KW ) |
| +1mm shift in $X$ | 0.32 | -75 | 57 | 105 | 50 | 2.4 (Total 51KW) |
| +0.5mrad angle in X | 0.32 | 245 | 120 | 90 | 50 | 3.0 (Total 54 KW ) |
| 20\% radiator thickness | 0.32 | -90 | 115 | 90 | 50 | 2.3 (Total 49KW) |

Check with a Fine Rectangular Grid for $\Delta Z=1 \mathrm{~m}$ absorber

| Nominal , 1m long in Z section | 0.32 | -90 | 135 | 86 | 50 | 2.9 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| $\sigma^{(x, y)}$ beam $=0.33 \mathrm{~mm}, 1 \mathrm{~m}$ long in Z | 0.32 | -90 | 140 | 120 | 55 | 8.0 |
| -1 mm shift in Y, 1m long in Z | 0.37 | -90 | 50 | 135 | 55 | 4.2 |

## Potential Problems and Mitigations (Vitaly)

- At larger vertical angles ( $500 \mu \mathrm{rad}$ ), the beam can penetrate deep into CPS passed the second magnet.
- Temperatures will be OK
- There might be radiation issues.
- Large angles should be prevented by an interlock on the photon beam position.

$$
\theta_{\mathrm{y}}=+0.5 \mathrm{mrad}
$$



- At lower beam position (-1mm), at horizontal offsets ( 1 mm ), or wider beam ( $\sigma=1.5 \mathrm{~mm}$ ), the hot spot is just before the first magnet
- There should not be high temperature issues.
- Radiation dose rate to the magnet might be elevated.
- The beam positions should be monitored and interlocked.
- The beam width needs to be measure on a regular basis with wire scans.



## KLCPS69 Test Summary (Pavel)

| Test Configuration Name | Hot Spot Location Section | $\mathrm{R}_{\max }(\mathrm{cm})$ | $\phi_{\max }(\mathrm{deg})$ | $\mathrm{Z}_{\text {max }}(\mathrm{cm})$ | $\mathrm{T}_{\text {max }}\left({ }^{\circ} \mathrm{C}\right)$ | $\mathrm{T}_{\text {cold }}\left({ }^{\circ} \mathrm{C}\right)$ | Maximum power (KW/cm ${ }^{3}$ ) |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| All Nominal ( $\sigma^{(x, y)}{ }_{\text {beam }}=1 \mathrm{~mm}, 4$ holes) | Keyhole | 0.04 | +90 | 37 | 200 | 55 | 7 |
| $\sigma^{(x, y)}{ }_{\text {beam }}=0.33 \mathrm{~mm}$ | Keyhole | 0.1 | +90 | 43 | 250 | 65 | 14 |
| $\sigma^{(x, y)}{ }_{\text {beam }}=1.5 \mathrm{~mm}$ | Keyhole | 0.2 | +90 | 8.5 | 205 | 55 | 5 |
| 97\% B-field | Circular | 0.15 | +90 | 58.5 | 205 | 60 | 8 |
| 103\% B-field | Keyhole | 0.1 | +90 | 33 | 200 | 55 | 7 |
| -1mm shift in $Y$ | Keyhole | 0.2 | +90 | 8 | 220 | 60 | 7 |
| +1mm shift in $Y$ | Circular | 0.1 | +90 | 57 | 225 | 60 | 6.5 |
| -0.5mrad angle in $Y$ | Keyhole | 0.2 | +90 | 8.5 | 220 | 60 | 6.5 |
| +0.5mrad angle in $Y$ | Circular | 0.15 | +90 | 58 | 235 | 60 | 7 |
| +1mm shift in X | Keyhole | 0.5 | +70 | 7.5 | 245 | 60 | 6 |
| +0.5mrad angle in $X$ | Keyhole | 0.45 | +70 | 8 | 250 | 60 | 6 |

Check with a Fine Rectangular Grid

| All Nominal $\left(\sigma^{(x, y)}\right.$ beam $=1 \mathrm{~mm}, 4$ holes $)$ | Keyhole | 0.13 | +90 | 37 | 230 | 100 | 7 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| $\sigma^{(x, y)}$ beam $=0.33 \mathrm{~mm}$ | Keyhole | 0.1 | +90 | 43 | 290 | 105 | 14 |
| +0.5 mrad angle in $Y$ | Circular | 0.15 | +90 | 58 | 275 | 105 | 7 |
| +1 mm shift in $X$ | Keyhole | 0.5 | +70 | 8.2 | 260 | 100 | 4 |

## Potential Problems and Mitigations (Pavel)

- At very large vertical angles ( 500 urad), the beam can penetrate deep into CPS and cause somewhat elevated temperatures ( $275^{\circ} \mathrm{C}$ ).
- The radiation environment is probably not going to be affected much.
- The photon beam position needs to be monitored and used in the beam interlock.
- At large horizontal shifts ( $\sim 1 \mathrm{~mm}$ ), the beam can impact the upstream wall of the absorber missing the keyhole and thus cause high temperatures ( $300^{\circ} \mathrm{C}$ ).
- The radiation environment is probably not going to be affected much.
- Beam position need to be monitored and beam needs to be shut off at large excursions.



## Thermal Deformations

- Assume uniform and isotropic copper absorber not attached to anything.

- Thermal strain for uniform isotropic body is $\varepsilon=\alpha \cdot \Delta T$.
- Assume that such a displacement actually occurs.
- With an approximation for the curvature radius $r \approx \frac{H}{\alpha \Delta T}$, and for the sagitta $s=r-\sqrt{r^{2}-\frac{L^{2}}{4}} \approx \frac{\alpha}{8} \cdot \Delta T \cdot \frac{L^{2}}{H}$, assuming $\alpha \approx 1.674 \cdot 10^{-5} \frac{1}{K}$, we get that:
a) At $L=94 \mathrm{~cm}, H=10 \mathrm{~cm}, \Delta T=150^{\circ} \mathrm{C}, s \approx 2.8 \mathrm{~mm}$.
- Sagitta is almost as large as a third of the photon beam channel!
b) At $L=229 \mathrm{~cm}, H=10 \mathrm{~cm}, \Delta T=100^{\circ} \mathrm{C}, s \approx 11.0 \mathrm{~mm}$.
- Sagitta is larger than the whole beam channel!
c) At $L=94 \mathrm{~cm}, H=5 \mathrm{~cm}, \Delta T=30^{\circ} \mathrm{C}, s \approx 1.1 \mathrm{~mm}$.
- We may need to have shorter than one-meter-long absorber segments along the beam.
- The temperature change over the long part of the absorber should be $\Delta T<30^{\circ} \mathrm{C}$.
d) At $L=10 \mathrm{~cm}, H=5 \mathrm{~cm}, \Delta T=500^{\circ} \mathrm{C}, s \approx 0.2 \mathrm{~mm}$.
- We may need to have $\sim 5 \mathrm{~cm}$ high and 10 cm long slits every 10 cm to avoid large deformation in the high temperature areas.
- These concepts needs to be properly modeled and calculated in ANSYS.


## Thermal stresses

- Assume uniform and isotropic copper absorber not attached to anything.
- Thermal strain for uniform isotropic body is $\varepsilon=\alpha \Delta T$.
- Thermal stresses for uniform isotropic body are
- $\sigma_{\text {norm }}=E \cdot \varepsilon=\alpha \cdot E \cdot \Delta T$ for normal stresses,
- $\sigma_{\text {sheer }}=G \cdot \varepsilon=\alpha \cdot G \cdot \Delta T$ for sheer stresses.
- For copper :
- $\sigma_{\text {max }}^{Y} \approx 283 \cdot 10^{6} \mathrm{~Pa}, \sigma_{\text {max }}^{T} \approx 350 \cdot 10^{6} \mathrm{~Pa} ; \alpha \approx 1.674 \cdot 10^{-5} \frac{1}{\mathrm{~K}} ; G \approx 4.4 \cdot 10^{10} \mathrm{~Pa}$ and $E \approx 12.6 \cdot 10^{10} \mathrm{~Pa}$.
- These constant depend on the type of the copper used.
- Using Tim's numbers, where available.
- Maximum allowed temperature differences for these numbers would be :
- $\Delta T=134^{\circ} \mathrm{C} / 166^{\circ} \mathrm{C}$ for normal stress,
- $\Delta T=384^{\circ} \mathrm{C} / 475^{\circ} \mathrm{C}$ for sheer stress.
- Both of our CPS models can avoid $\Delta T=384^{\circ} \mathrm{C}$ temperature differences across the absorber by monitoring and controlling the beam conditions.
- Normal thermal stresses are not expected to be large
- There may be nothing we need to do to avoid excessive thermal stresses in the CPS models if there are no compression stresses involved.
- This needs to be checked with ANSYS realistic model.
- Mechanical stresses can be induced.
- Normal stresses can also be present.
- Even the presence of excessive stresses does not mean failure
- CPS mechanical models need to be solved to determine the behavior of the absorber at given temperature and boundary conditions.


## Conclusions

- Both models provide acceptable temperatures assuming care is taken when designing the absorber.
- Pavel's model $\mathrm{T}_{\max } \approx 300^{\circ} \mathrm{C}$ for reasonably possible beam conditions.
- Vitaly's model $\mathrm{T}_{\max } \approx 150^{\circ} \mathrm{C}$ for reasonably possible beam conditions.
- Deformations and thermal stresses are highly unlikely to be serious problems.
- Proper design for the absorber and mounting will be needed.
- Radiation is another environment is another important criteria for the CPS design
- The desired goal is to have PDE on the level of $25 \mathrm{rem} / \mathrm{h}$ in the tagger hall, as indicated in the PAC proposal.
- This is not a new goal, see e-mail from December 7, 2022, in the JLAB "mailman" archive.
- https://mailman.jlab.org/pipermail/halld-cps/2022-December/000004.html
- Activation dose after 1000-hour continuous beam operations and 1 hour break needs to be low enough for a controlled access into the hall.
- I would like to decide on the CPS model before next Monday meeting.
- Still need FLUKA data on beam size and beam background.
- Need to have the material weights for Tim to estimate the cost for models.

